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INTRODUCTION

Investigate elements of simple mechanical power transmission systems.

- Simple systems:
 - Parameters of simple and compounded geared systems.
 - Efficiency of lead screws and screw jacks.
- Couplings and energy storage:
 - Universal couplings and conditions for constant-velocity.
 - Importance of energy storage elements and their applications.

GUIDANCE

This document is prepared to break the unit material down into bite size chunks. You will see the learning outcomes above treated in their own sections. Therein you will encounter the following structures;





1.2 Simple Systems

1.2.1 Belt Drive Dynamics

Purpose

Understand flat and v-section belts; limiting coefficient friction; limiting slack and tight side tensions; initial tension requirements; maximum power transmitted

Theory Revision

Consider the basic belt drive arrangement below...



The belt drive arrangement in *Figure 1* has the driving side on the left and the driven side on the right. This causes the bottom of the belt to be the tight side and the top to be the slack side. Some parameters of note in this arrangement...

- **D**₁, **D**₂ Respective diameters of the pulleys
- r₁, r₂ Respective radii of the pulleys
- ω_1, ω_2 Angular velocity of each pulley (in radians per second)
- θ_1, θ_2 Lap angle (angle subtended to centre of pulley by contact length of belt with pulley surface)
- T₁, T₂ Belt tensions on tight side and slack side, respectively
- S Belt length which does not touch the pulleys
- **C** Distance between pulley centres
- v₂ Linear belt velocity

Many formulae may be derived for such a pulley system, but the most important ones are ...

Belt lap angles for pulleys

$$\vartheta_{1} = 180^{o} - 2 \sin^{-1} \left\{ \frac{D_{2} - D_{1}}{2C} \right\}$$
(Eq 3)
$$\vartheta_{2} = 180^{o} + 2 \sin^{-1} \left\{ \frac{D_{2} - D_{1}}{2C} \right\}$$
(Eq 4)



Power transmitted by the system

$$P = T_1 (1 - e^{-\mu\theta}) v \tag{Eq 5}$$

where: -

 $v = r\omega$

 $\boldsymbol{\mu}$ is the coefficient of friction between the belt and the pulley

Belt pitch length

$$L = 2C + 1.57(D_2 - D_1) + \frac{(D_2 - D_1)^2}{4C}$$

Span length

$$S = \sqrt{C^2 - \left[\frac{D_2 - D_1}{2}\right]^2}$$

Worked Example 2



A flat belt drive system consists of two parallel pulleys of diameter 300 and 500 mm, which have a distance between centres of 600 mm. Given that the maximum belt tension is not to exceed 1.5 kN, the coefficient of friction between the belt and pulley is 0.3 and the larger pulley rotates at 40 rad/sec. Find;

- a) the belt lap angles for the pulleys
- b) the power transmitted by the system
- c) the belt pitch length L
- d) the pulley system span length between centres

ANSWERS

(a)

The belt lap angles for the pulleys are given by equations (Eq 3) and (Eq 4) ...

$$\vartheta_1 = 180^o - 2\sin^{-1}\left\{\frac{D_2 - D_1}{2C}\right\} = 180^o - 2\sin^{-1}\left\{\frac{0.5 - 0.3}{2(0.6)}\right\} = 160.8^o$$

$$\vartheta_2 = 180^o + 2\sin^{-1}\left\{\frac{D_2 - D_1}{2C}\right\} = 180^o + 2\sin^{-1}\left\{\frac{0.5 - 0.3}{2(0.6)}\right\} = 199.2^o$$

(b)

The power transmitted by the system is given by equation (Eq 5) ...

$$P = T_1 \big(1 - e^{-\mu\theta} \big) v$$

where: -

 $v = r\omega = (0.25)(40) = 10 \ m. \ s^{-1}$

The angle θ_1 is expressed in degrees in part (a) but we must convert this to radians to be compatible with equation (Eq 5) ...



$$160.8^{o} \equiv \left(160.8 \times \frac{\pi}{180}\right) radians = 2.81 rads.$$

$$\therefore P = T_{1} \left(1 - e^{-\mu\theta}\right) v = 1500 \left(1 - e^{-(0.3)(2.81)}\right) (10) = 8.54 kW$$
(c)

The belt pitch length is given by equation (Eq 6) ...

$$L = 2C + 1.57(D_2 - D_1) + \frac{(D_2 - D_1)^2}{4C} = 2(0.6) + 1.57(0.5 - 0.3) + \frac{(0.5 - 0.3)^2}{4(0.6)} = 1.53 m$$

(d)

Pulley system span length is given by equation (Eq 7) ...

$$S = \sqrt{C^2 - \left[\frac{D_2 - D_1}{2}\right]^2} = \sqrt{0.6^2 - \left[\frac{0.5 - 0.3}{2}\right]^2} = 0.59 m$$

Additional worked examples are available in the eBooks section on Moodle.

1.2.2 Friction Clutches

Purpose

Understand flat single and multi-plate clutches; conical clutches; coefficient of friction; spring force requirements; maximum power transmitted by constant wear and constant pressure theories; validity of theories

Theory Revision

Consider the friction clutch shown below .



