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INTRODUCTION

Analyse natural and damped vibrations within translational and rotational mass-spring systems

- Types of motion:
 - Simple harmonic motion.
 - Natural frequency of vibration in mass-spring systems.
- Damped systems:
 - Frequency of damped vibrations in mass-spring-damper systems.
 - The conditions for an external force to produce resonance.

GUIDANCE

This document is prepared to break the unit material down into bite size chunks. You will see the learning outcomes above treated in their own sections. Therein you will encounter the following structures;





Types of motion: 1

1.1 Simple harmonic motion.

A particle is said to be under Simple Harmonic Motion (SHM) if its acceleration along a line is directly proportional to its displacement from a fixed point on that line.

Consider the motion of a particle A, rotating in a circle with a constant angular velocity ω , as shown in Figure 1 (a).

time' (b) Figure 1 Simple Harmonic Motion Consider now the vertical displacement of A from the x-axis, as shown by the distance y_c. If P is rotating at a constant angular velocity ω then the periodic time τ to travel an angular distance of 2π , is given by: $\tau = \frac{2\pi}{2\pi}$ (Eq 1) Let f = frequency of motion C (in Hertz), where $f = \frac{1}{t} = \frac{\omega}{2\pi}$ (Eq 2) To determine whether SHM is taking place, consider the motion of A in the vertical direction (y-axis). Now $y_c = OA \sin \omega t$, i.e.,

 $y = r \sin \omega t$, where t = time in seconds (Eq 3) Plotting of equation (Eq 3) against t results in the sinusoidal variation for displacement, as shown in Figure 1 (b).

We know that $\mathbf{v}_{A} = \boldsymbol{\omega} \mathbf{r}$, which is the tangential velocity of the particle A. From the velocity vector diagram, at the point A on the circle of Figure 1 (a),

$$v_c = v_A \cos \theta = v_A \cos \omega t$$

Plotting of equation (Eq 4) against t results in the sinusoidal variation for the velocity v_c, as shown in Figure 1 (b).

The centripetal acceleration of A = $a_A = \omega^2 r$

Now
$$a_c = -a_A \sin \theta$$
, $\therefore a_c = -\omega^2 r \sin \omega t$ (Eq 5)

Plotting equation (Eq 5) against t results in the sinusoidal variation for the acceleration at C, a_c, as shown in Figure 1 (b).

Substituting equation (Eq 3) into equation (Eq 5) gives:

 $a_c = -\omega^2 y_c$





(Eq 4)



Equation (Eq 6) shows that the acceleration along the y-axis is directly proportional to the displacement along this line, therefore the point C is moving with SHM. Now,

$$T = \frac{2\pi}{\omega}, \text{ but from equation (Eq 6)} \qquad a_C = -\omega^2 y_C \qquad \text{i.e.} \qquad \omega^2 = \frac{a}{y}$$

Therefore, $T = \frac{2\pi}{\sqrt{\frac{a}{y}}} \qquad \text{or} \qquad T = 2\pi \sqrt{\frac{y}{a}} \qquad \text{i.e.} \qquad T = 2\pi \sqrt{\frac{displacemet}{acceleration}}$

In general, from equation (Eq 6) $a + \omega^2 y = 0$

(Eq 7)

1.2 Natural frequency of vibration in mass-spring systems.

1.2.1 The Spring-Mass System vibrating horizontally

Consider a mass m resting on a smooth surface and attached to a spring of stiffness k, as shown in Figure 2.



1.2.1 The Spring-Mass System vibrating horizontally

Consider a mass m, supported by a vertical spring of stiffness k, as shown in Figure 3. In this equilibrium position, the mass has an initial downward static deflection of y_0 . If the mass is given an additional downward displacement of y and then released, it will vibrate vertically.

